NFPA 291

Recommended Practice for Fire Flow Testing and Marking of Hydrants

2007 Edition



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NFPA 291

Recommended Practice for

Fire Flow Testing and Marking of Hydrants

2007 Edition

This edition of NFPA 291, *Recommended Practice for Fire Flow Testing and Marking of Hydrants*, was prepared by the Technical Committee on Private Water Supply Piping Systems and released by the Technical Correlating Committee on Automatic Sprinkler Systems. It was issued by the Standards Council on July 28, 2006, with an effective date of August 17, 2006, and supersedes all previous editions.

This edition of NFPA 291 was approved as an American National Standard on August 17, 2006.

Origin and Development of NFPA 291

The NFPA Committee on Public Water Supplies for Private Fire Protection presented the idea of indicating the relative available fire service water supply from hydrants in its 1934 report. The Committee felt then and feels now that such an indication is of substantial value to water and fire departments. The following recommendations were initially adopted in 1935. The Committee agreed that tests of individual hydrants did not give as complete and satisfactory results as group testing but expressed the opinion that tests of individual hydrants did have sufficient value to make the following recommendations worthy of adoption. This was reconfirmed with minor editorial changes in 1974.

The 1977 edition was completely rewritten and a chapter on the flow testing of hydrants was added.

The 1982 edition was reconfirmed by the Committee. The 1988 edition of the document noted several changes that clarified and reinforced certain recommendations. Specific guidance was added on the correct method of utilizing a pitot tube to gain accurate test results.

The 1995 edition incorporated several changes in an attempt to make the document more user friendly. Changes were also incorporated with regard to the layout of hydrant and water flow tests.

The 2002 edition clarified the recommendations for flow tests and was restructured to comply with the *Manual of Style for NFPA Technical Committee Documents*.

The 2007 edition represents a reconfirmation of the 2002 edition, as there are no technical changes.

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Committee Scope: This Committee shall have the primary responsibility for documents on private piping systems supplying water for fire protection and for hydrants, hose houses, and valves. The Committee is also responsible for documents on fire flow testing and marking of hydrants.

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Recommended Practice for

Fire Flow Testing and Marking of Hydrants

2007 Edition

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NOTICE: An asterisk (*) following the number or letter designating a paragraph indicates that explanatory material on the paragraph can be found in Annex A.

Information on referenced publications can be found in Chapter 2.

Chapter 1 Administration

1.1 Scope. The scope of this document is fire flow testing and marking of hydrants.

1.2 Purpose. Fire flow tests are conducted on water distribution systems to determine the rate of flow available at various locations for fire-fighting purposes.

1.3 Application. A certain residual pressure in the mains is specified at which the rate of flow should be available. Additional benefit is derived from fire flow tests by the indication of possible deficiencies, such as tuberculation of piping or closed valves or both, which could be corrected to ensure adequate fire flows as needed.

1.4 Units. Metric units of measurement in this recommended practice are in accordance with the modernized metric system known as the International System of Units (SI). Two units (liter and bar), outside of but recognized by SI, are commonly used in international fire protection. These units are listed in Table 1.4 with conversion factors.

1.4.1 If a value for measurement as given in this recommended practice is followed by an equivalent value in other units, the first value stated is to be regarded as the recommendation. A given equivalent value might be approximate.

Table 1.4 SI Units and Conversion Factors

Unit Name	Unit Symbol	Conversion Factor
liter	L	1 gal = 3.785 L
liter per minute per square meter	$(L/min)/m^2$	1 gpm ft ² = (40.746 L/min)/m ²
cubic decimeter	dm^3	$1 \text{ gal} = 3.785 \text{ dm}^3$
pascal	Pa	1 psi = 6894.757 Pa
bar	bar	1 psi = 0.0689 bar
bar	bar	$1 \text{ bar} = 10^5 \text{ Pa}$

Note: For additional conversions and information, see IEEE/ASTM-SI-10, Standard for Use of the International System of Units (SI): The Modern Metric System, 1992.

Chapter 2 Referenced Publications

2.1 General. The documents or portions thereof listed in this chapter are referenced within this recommended practice and should be considered part of the recommendations of this document.

2.2 NFPA Publications. (Reserved)

2.3 Other Publications.

2.3.1 IEEE Publications. Institute of Electrical and Electronics Engineers, Three Park Avenue, 17th Floor, New York, NY 10016-5997.

IEEE/ASTM-SI-10, Standard for Use of the International System of Units (SI): The Modern Metric System, 1992.

2.3.2 Other Publications. *Merriam-Webster's Collegiate Dictionary*, 11th edition, Merriam-Webster, Inc., Springfield, MA, 2003.

2.4 References for Extracts in Recommendations Sections. (Reserved)

Chapter 3 Definitions

3.1 General. The definitions contained in this chapter apply to the terms used in this recommended practice. Where terms are not defined in this chapter or within another chapter, they should be defined using their ordinarily accepted meanings within the context in which they are used. *Merriam-Webster's Collegiate Dictionary*, 11th edition, is the source for the ordinarily accepted meaning.

3.2 NFPA Official Definitions.

3.2.1* Authority Having Jurisdiction (AHJ). An organization, office, or individual responsible for enforcing the requirements of a code or standard, or for approving equipment, materials, an installation, or a procedure.

3.2.2* Listed. Equipment, materials, or services included in a list published by an organization that is acceptable to the authority having jurisdiction and concerned with evaluation of products or services, that maintains periodic inspection of production of listed equipment or materials or periodic evaluation of services, and whose listing states that either the equipment, material, or service meets appropriate designated standards or has been tested and found suitable for a specified purpose.

3.2.3 Should. Indicates a recommendation or that which is advised but not required.

3.3 General Definitions.

3.3.1 Rated Capacity. The flow available from a hydrant at the designated residual pressure (rated pressure), either measured or calculated.

3.3.2 Residual Pressure. The pressure that exists in the distribution system, measured at the residual hydrant at the time the flow readings are taken at the flow hydrants.

3.3.3 Static Pressure. The pressure that exists at a given point under normal distribution system conditions measured at the residual hydrant with no hydrants flowing.



Chapter 4 Flow Testing

4.1 Rating Pressure.

4.1.1 For the purpose of uniform marking of fire hydrants, the ratings should be based on a residual pressure of 20 psi (1.4 bar) for all hydrants having a static pressure in excess of 40 psi (2.8 bar).

4.1.2 Hydrants having a static pressure of less than 40 psi (2.8 bar) should be rated at one-half of the static pressure.

4.1.3 It is generally recommended that a minimum residual pressure of 20 psi (1.4 bar) should be maintained at hydrants when delivering the fire flow. Fire department pumpers can be operated where hydrant pressures are less, but with difficulty.

4.1.4 Where hydrants are well distributed and of the proper size and type (so that friction losses in the hydrant and suction line are not excessive), it might be possible to set a lesser pressure as the minimum pressure.

4.1.5 A primary concern should be the ability to maintain sufficient residual pressure to prevent developing a negative pressure at any point in the street mains, which could result in the collapse of the mains or other water system components or back-siphonage of polluted water from some other interconnected source.

4.1.6 It should be noted that the use of residual pressures of less than 20 psi (1.4 bar) is not permitted by many state health departments.

4.2 Procedure.

4.2.1 Tests should be made during a period of ordinary demand.

4.2.2 The procedure consists of discharging water at a measured rate of flow from the system at a given location and observing the corresponding pressure drop in the mains.

4.3 Layout of Test.

4.3.1 After the location where the test is to be run has been determined, a group of test hydrants in the vicinity is selected.

4.3.2 Once selected, due consideration should be given to potential interference with traffic flow patterns, damage to surroundings (e.g., roadways, sidewalks, landscapes, vehicles, and pedestrians), and potential flooding problems both local and remote from the test site.

4.3.3 One hydrant, designated the residual hydrant, is chosen to be the hydrant where the normal static pressure will be observed with the other hydrants in the group closed, and where the residual pressure will be observed with the other hydrants flowing.

4.3.4 This hydrant is chosen so it will be located between the hydrant to be flowed and the large mains that constitute the immediate sources of water supply in the area. In Figure 4.3.4, test layouts are indicated showing the residual hydrant designated with the letter R and hydrants to be flowed with the letter F.

4.3.5 The number of hydrants to be used in any test depends upon the strength of the distribution system in the vicinity of the test location.



Arrows indicate direction of flow: R - residual hydrant; F - flow hydrant

FIGURE 4.3.4 Suggested Test Layout for Hydrants.

4.3.6 To obtain satisfactory test results of theoretical calculation of expected flows or rated capacities, sufficient discharge should be achieved to cause a drop in pressure at the residual hydrant of at least 25 percent, or to flow the total demand necessary for fire-fighting purposes.

4.3.7 If the mains are small and the system weak, only one or two hydrants need to be flowed.

4.3.8 If, on the other hand, the mains are large and the system strong, it may be necessary to flow as many as seven or eight hydrants.

4.4 Equipment.

4.4.1 The equipment necessary for field work consists of the following:

- A single 200 psi (14 bar) bourdon pressure gauge with 1 psi (0.0689 bar) graduations.
- (2) A number of pitot tubes.
- (3) Hydrant wrenches.
- (4) 50 or 60 psi (3.5 or 4.0 bar) bourdon pressure gauges with 1 psi (0.0689 bar) graduations, and scales with ½6 in. (1.6 mm) graduations [One pitot tube, a 50 or 60 psi (3.5 or 4.0 bar) gauge, a hydrant wrench, a scale for each hydrant to be flowed].
- (5) A special hydrant cap tapped with a hole into which a short length of ¼ in. (6.35 mm) brass pipe is fitted; this pipe is provided with a T connection for the 200 psi (14 bar) gauge and a cock at the end for relieving air pressure.

4.4.2 All pressure gauges should be calibrated at least every 12 months, or more frequently depending on use.

4.4.3 When more than one hydrant is flowed, it is desirable and could be necessary to use portable radios to facilitate communication between team members.

4.4.4 It is preferred to use stream straightener with a known coefficient of discharge when testing hydrants due to a more streamlined flow and more accurate pitot reading.



4.5 Test Procedure.

4.5.1 In a typical test, the 200 psi (14 bar) gauge is attached to one of the $2\frac{1}{2}$ in. (6.4 cm) outlets of the residual hydrant using the special cap.

4.5.2 The cock on the gauge piping is opened, and the hydrant valve is opened full.

4.5.3 As soon as the air is exhausted from the barrel, the cock is closed.

4.5.4 A reading (static pressure) is taken when the needle comes to rest.

4.5.5 At a given signal, each of the other hydrants is opened in succession, with discharge taking place directly from the open hydrant butts.

4.5.6 Hydrants should be opened one at a time.

4.5.7 With all hydrants flowing, water should be allowed to flow for a sufficient time to clear all debris and foreign substances from the stream(s).

4.5.8 At that time, a signal is given to the people at the hydrants to read the pitot pressure of the streams simultaneously while the residual pressure is being read.

4.5.9 The final magnitude of the pressure drop can be controlled by the number of hydrants used and the number of outlets opened on each.

4.5.10 After the readings have been taken, hydrants should be shut down slowly, one at a time, to prevent undue surges in the system.

4.6 Pitot Readings.

4.6.1 When measuring discharge from open hydrant butts, it is always preferable from the standpoint of accuracy to use $2\frac{1}{2}$ in. (6.4 cm) outlets rather than pumper outlets.

4.6.2 In practically all cases, the $2\frac{1}{2}$ in. (6.4 cm) outlets are filled across the entire cross-section during flow, while in the case of the larger outlets there is very frequently a void near the bottom.

4.6.3 When measuring the pitot pressure of a stream of practically uniform velocity, the orifice in the pitot tube is held downstream approximately one-half the diameter of the hydrant outlet or nozzle opening, and in the center of the stream.

4.6.4 The center line of the orifice should be at right angles to the plane of the face of the hydrant outlet.

4.6.5 The air chamber on the pitot tube should be kept elevated.

4.6.6 Pitot readings of less than 10 psi (0.7 bar) and more than 30 psi (2.0 bar) should be avoided, if possible.

4.6.7 Opening additional hydrant outlets will aid in controlling the pitot reading.

4.6.8 With dry barrel hydrants, the hydrant valve should be wide open to minimize problems with underground drain valves.

4.6.9 With wet barrel hydrants, the valve for the flowing outlet should be wide open to give a more streamlined flow and a more accurate pitot reading. *(See Figure 4.6.9.)*



FIGURE 4.6.9 Pitot Tube Position.

4.7 Determination of Discharge.

4.7.1 At the hydrants used for flow during the test, the discharges from the open butts are determined from measurements of the diameter of the outlets flowed, the pitot pressure (velocity head) of the streams as indicated by the pitot gauge readings, and the coefficient of the outlet being flowed as determined from Figure 4.7.1.



FIGURE 4.7.1 Three General Types of Hydrant Outlets and Their Coefficients of Discharge.

4.7.2 If flow tubes (stream straighteners) are being utilized, a coefficient of 0.95 is suggested unless the coefficient of the tube is known.

4.7.3 The formula used to compute the discharge, Q_i in gpm from these measurements is as follows:

$$Q = 29.84cd^2 \sqrt{p}$$
 (4.7.3)

where:

c = coefficient of discharge (see Figure 4.7.1)

d = diameter of the outlet in inches

p = pitot pressure (velocity head) in psi

4.8 Use of Pumper Outlets.

4.8.1 If it is necessary to use a pumper outlet, and flow tubes (stream straighteners) are not available, the best results are obtained with the pitot pressure (velocity head) maintained between 5 psi and 10 psi (0.3 bar and 0.7 bar).

4.8.2 For pumper outlets, the approximate discharge can be computed from Equation 4.7.3 using the pitot pressure (velocity head) at the center of the stream and multiplying the result by one of the coefficients in Table 4.8.2, depending upon the pitot pressure (velocity head).

4.8.3 These coefficients are applied in addition to the coefficient in Equation 4.7.3 and are for average-type hydrants.

4.9 Determination of Discharge Without a Pitot.

into a hydrant cap can be used.

outlet at the same elevation.

4.10 Calculation Results.

use of Equation 4.7.3.

stream, are approximately the same.

4.9.1 If a pitot tube is not available for use to measure the hydrant discharge, a 50 or 60 psi (3.5 or 4.0 bar) gauge tapped

4.9.2 The hydrant cap with gauge attached is placed on one outlet, and the flow is allowed to take place through the other

4.9.3 The readings obtained from a gauge so located, and the readings obtained from a gauge on a pitot tube held in the

4.10.1 The discharge in gpm (L/min) for each outlet flowed is obtained from Table 4.10.1 (a) and Table 4.10.1 (b) or by the

Pitot I (Veloc	Pitot Pressure (Velocity Head)			
psi	bar	Coefficient		
2	0.14	0.97		
3	0.21	0.92		
4	0.28	0.89		
5	0.35	0.86		
6	0.41	0.84		
7 and over	0.48 and over	0.83		

Table 4.8.2 Pumper Outlet Coefficients

Table	4.10.1(a)	Theoretica	l Discharge	Through	Circular	Orifices
(U.S.	Gallons of	f Water per	Minute)			

Orifice Size (in.) Pitot Velocity Pressure* Discharge 2.625 (psi) Feet[†] (ft/sec) 2.25 2.375 2.5 2.75 3.25 3.5 3.75 4.5 12.20 2.31 4.61 17.25 6.92 21.13 9.23 24.39 27.26 11.54 13.84 99.87 16.15 32.26 18.46 34.49 20.76 36.58 23.07 38 56 25.38 40.45 27.68 42.24 29.99 43 97 32.30 45.63 34.61 47.22 36.91 48.78 39.99 50.28 41.53 51.73 43.83 53.15 46.14 54.54 50.7557.19 55.37 59.74 59.98 62.18 64.60 64.52 69.21 66.79 73.82 68.98 78.44 71.10 83.05 73.16 87.67 75.17 99 98 77.11 96.89 79.03 101.51 80.88 106.12 82.70 84.48 110.74 115.3586.22

Table 4.10.1(a) Continued

		Orifice Size (in.)												
Pitot Pressure*		Velocity Discharge												
(psi)	Feet+	(ft/sec)	2	2.25	2.375	2.5	2.625	2.75	3	3.25	3.5	3.75	4	4.5
52 54 56 58 60	119.96 124.58 129.19 133.81 138.42	87.93 89.61 91.20 92.87 94.45	861 877 893 909 925	1089 1110 1130 1150 1170	1214 1237 1260 1282 1304	1345 1370 1396 1420 1445	$ \begin{array}{r} 1483 \\ 1511 \\ 1539 \\ 1566 \\ 1593 \\ \end{array} $	1627 1658 1689 1719 1748	1937 1974 2010 2045 2080	$2273 \\ 2316 \\ 2359 \\ 2400 \\ 2441$	2636 2686 2735 2784 2831	3026 3084 3140 3196 3250	3443 3508 3573 3636 3698	4357 4440 4522 4602 4681
62 64 66 68 70	$\begin{array}{c} 143.03 \\ 147.65 \\ 152.26 \\ 156.88 \\ 161.49 \end{array}$	96.01 97.55 99.07 100.55 102.03	940 955 970 984 999	1189 1209 1227 1246 1264	1325 1347 1367 1388 1408	1469 1492 1515 1538 1560	1619 1645 1670 1696 1720	1777 1805 1833 1861 1888	2115 2148 2182 2215 2247	2482 2521 2561 2599 2637	2878 2924 2970 3014 3058	3304 3357 3409 3460 3511	3759 3820 3879 3937 3995	4758 4834 4909 4983 5056
72 74 76 78 80	$166.10 \\ 170.72 \\ 175.33 \\ 179.95 \\ 184.56$	$103.47 \\104.90 \\106.30 \\107.69 \\109.08$	1013 1027 1041 1054 1068	1282 1300 1317 1334 1351	1428 1448 1467 1487 1505	$1583 \\ 1604 \\ 1626 \\ 1647 \\ 1668$	1745 1769 1793 1816 1839	1915 1941 1967 1993 2018	2279 2310 2341 2372 2402	2674 2711 2748 2784 2819	3102 3144 3187 3228 3269	3561 3610 3658 3706 3753	4051 4107 4162 4217 4270	5127 5198 5268 5337 5405
82 84 86 88 90	189.17 193.79 198.40 203.02 207.63	$110.42 \\111.76 \\113.08 \\114.39 \\115.68$	1081 1094 1107 1120 1132	1368 1385 1401 1417 1433	1524 1543 1561 1579 1597	1689 1709 1730 1750 1769	1862 1885 1907 1929 1951	2043 2068 2093 2117 2141	2432 2461 2491 2519 2548	2854 2889 2923 2957 2990	3310 3350 3390 3429 3468	3800 3846 3891 3936 3981	4323 4376 4428 4479 4529	5472 5538 5604 5668 5733
92 94 96 98 100	212.24 216.86 221.47 226.09 230.70	116.96 118.23 119.48 120.71 121.94	1145 1157 1169 1182 1194	1449 1465 1480 1495 1511	1614 1632 1649 1666 1683	1789 1808 1827 1846 1865	$1972 \\1994 \\2015 \\2035 \\2056$	2165 2188 2211 2234 2257	$2576 \\ 2604 \\ 2631 \\ 2659 \\ 2686$	3023 3056 3088 3120 3152	$3506 \\ 3544 \\ 3582 \\ 3619 \\ 3655$	4025 4068 4111 4154 4196	4579 4629 4678 4726 4774	5796 5859 5921 5982 6043
102 104 106 108 110	$235.31 \\ 239.93 \\ 244.54 \\ 249.16 \\ 253.77$	$123.15 \\124.35 \\125.55 \\126.73 \\127.89$	1205 1217 1229 1240 1252	1526 1541 1555 1570 1584	1700 1716 1733 1749 1765	1884 1902 1920 1938 1956	2077 2097 2117 2137 2157	2279 2301 2323 2345 2367	2712 2739 2765 2791 2817	3183 3214 3245 3275 3306	3692 3728 3763 3799 3834	4238 4279 4320 4361 4401	4822 4869 4916 4962 5007	6103 6162 6221 6280 6338
112 114 116 118 120	$\begin{array}{c} 258.38 \\ 263.00 \\ 267.61 \\ 272.23 \\ 276.84 \end{array}$	129.05 130.20 131.33 132.46 133.57	1263 1274 1286 1297 1308	$ 1599 \\ 1613 \\ 1627 \\ 1641 \\ 1655 $	1781 1797 1813 1828 1844	1974 1991 2009 2026 2043	2176 2195 2215 2234 2252	2388 2409 2430 2451 2472	2842 2867 2892 2917 2942	3336 3365 3395 3424 3453	3869 3903 3937 3971 4004	4441 4480 4519 4558 4597	5053 5098 5142 5186 5230	$6395 \\ 6452 \\ 6508 \\ 6564 \\ 6619$
122 124 126 128 130	281.45 286.07 290.68 295.30 299.91	134.69 135.79 136.88 137.96 139.03	1318 1329 1340 1350 1361	1669 1682 1696 1709 1722	1859 1874 1889 1904 1919	2060 2077 2093 2110 2126	2271 2290 2308 2326 2344	2493 2513 2533 2553 2573	2966 2991 3015 3038 3062	3481 3510 3538 3566 3594	4038 4070 4103 4136 4168	4635 4673 4710 4748 4784	5273 5317 5359 5402 5444	6674 6729 6783 6836 6890
132 134 136	304.52 309.14 313.75	140.10 141.16 142.21	1371 1382 1392	1736 1749 1762	1934 1948 1963	2143 2159 2175	2362 2380 2398	2593 2612 2632	3086 3109 3132	3621 3649 3676	4200 4231 4263	4821 4858 4894	5485 5527 5568	6942 6995 7047

Notes:

(1) This table is computed from the formula $Q = 29.84cd^2\sqrt{p}$, with c = 1.00. The theoretical discharge of seawater, as from fireboat nozzles, can be found by subtracting 1 percent from the figures in Table 4.10.2.1, or from the formula $Q = 29.84cd^2\sqrt{p}$.

(2) Appropriate coefficient should be applied where it is read from hydrant outlet. Where more accurate (2) Appropriate coefficient should be appred where it is read noin hydraut outer, where more actuate results are required, a coefficient appropriate on the particular nozzle must be selected and applied to the figures of the table. The discharge from circular openings of sizes other than those in the table can readily be computed by applying the principle that quantity discharged under a given head varies as the square of the diameter of the opening.

*This pressure corresponds to velocity head.

 ± 1 psi = 2.307 ft of water. For pressure in bars, multiply by 0.01.

								Orific	e Size	(mm)				
Pitot		Velocity		<u> </u>			Γ							
(kPa)	Meters†	(m/sec)	51	57	60	64	67	70	76	83	89	95	101	114
6.89	0.70	3.72	455	568	629	716	785	857	1010	1204	1385	1578	1783	2272
13.8	1.41	5.26	644	804	891	1013	1111	1212	1429	1704	1960	2233	2524	3215
20.7	2.11	6.44	788	984	1091	1241	1360	1485	1750	2087	2400	2735	3091	3938
27.6	2.81	7.43	910	1137	1260	1433	1571	1714	2021	2410	2771	3158	3569	4547
34.5	3.52	8.31	1017	1271	1408	1602	1756	1917	2259	2695	3099	3530	3990	5084
41.4	4.22	9.10	1115	1392	1543	1755	1924	2100	2475	2952	3394	3867	4371	5569
48.3	4.92	9.83	1204	1504	1666	1896	2078	2268	2673	3189	3666	4177	4722	6015
55.2	5.63	10.51	1287	1608	1781	2027	2221	2425	2858	3409	3919	4466	5048	6431
62.0	6.33	11.15	1364	1704	1888	2148	2354	2570	3029	3613	4154	4733	5349	6815
68.9	7.03	11.75	1438	1796	1990	2264	2482	2709	3193	3808	4379	4989	5639	7184
75.8	7.73	12.33	1508	1884	2087	2375	2603	2841	3349	3995	4593	5233	5915	7536
82.7	8.44	12.87	1575	1968	2180	2481	2719	2968	3498	4172	4797	5466	6178	7871
89.6	9.14	13.40	1640	2048	2270	2582	2830	3089	3641	4343	4994	5690	6431	8193
96.5	9.84	13.91	1702	2120	2355	2080	2937	3200	3779	4507	5182	5905	6074	8503
105	10.55	14.39	1758	2190	2433	2709	3136	3312	3904	4030	5522	6204	0895	0079
117	11.25	14.07	1874	2209	2515	2001	3934	3423	4055	4012	5555	6509	7123	9078
194	12.66	15.55	1074	2341	2595	2951	3294	3634	4101	5100	5874	6603	7565	9502
121	13.36	16.20	1983	9477	2010	3199	3499	3735	4403	5951	6038	6880	7776	9006
138	14.06	16.69	2035	2549	2817	3905	3512	3834	4519	5390	6197	7061	7981	10168
152	15.47	17.43	2136	2668	2956	3363	3686	4023	4743	5657	6504	7410	8376	10671
165	16.88	18.21	2225	2779	3080	3504	3840	4192	4941	5893	6776	7721	8727	11118
179	18.28	18.95	2318	2895	3208	3650	4000	4366	5147	6138	7058	8042	9090	11580
193	19.69	19.67	2407	3006	3331	3790	4153	4534	5344	6374	7329	8350	9438	12024
207	21.10	20.36	2492	3113	3450	3925	4301	4695	5535	6601	7590	8648	9775	12453
221	22.50	21.03	2575	3217	3564	4055	4444	4851	5719	6821	7842	8935	10100	12867
234	23.91	21.67	2650	3310	3668	4173	4573	4992	5884	7018	8070	9195	10393	13240
248	25.31	22.30	2728	3408	3776	4296	4708	5139	6058	7225	8308	9466	10699	13630
262	26.72	22.91	2804	3502	3881	4416	4839	5282	6227	7426	8539	9729	10997	14010
276	28.13	23.50	2878	3595	3983	4532	4967	5422	6391	7622	8764	9986	11287	14379
290	29.53	24.09	2950	3685	4083	4646	5091	5557	6551	7813	8984	10236	11570	14740
303	30.94	24.65	3015	3767	4173	4748	5204	5681	6696	7986	9183	10463	11826	15066
317	32.35	25.21	3084	3853	4269	4857	5323	5810	6849	8169	9393	10702	12096	15410
331	33.75	25.75	3152	3937	4362	4963	5439	5937	6999	8347	9598	10935	12360	15747
345	35.16	26.28	3218	4019	4453	5067	5553	6061	7145	8522	9799	11104	12619	16077
338 979	30.57	20.80	3278	4094	4550	5101	5057	6904	7410	8840	9901	11575	12000	16604
372	37.97	27.31	2402	4175	4024	5260	5700	6419	7419	0014	10175	11995	13104	17005
400	40 78	27.80	3465	4291	4705	5456	5074	6597	7694	9176	10551	19091	13588	17005
414	49 19	28.51	3595	4403	4878	5551	6083	6640	7897	9335	10734	19930	13893	17611
497	43.60	29.26	3580	4471	4954	5637	6178	6743	7949	9481	10901	12420	14039	17885
441	45.00	29.73	3638	4544	5035	5729	6278	6853	8078	9635	11078	12622	14267	18176
455	46.41	30.20	3695	4616	5114	5819	6377	6961	8206	9787	11253	12821	14492	18462
469	47.82	30.65	3751	4686	5192	5908	6475	7067	8331	9936	11425	13017	14713	18744
483	49.22	31.10	3807	4756	5269	5995	6570	7172	8454	10083	11594	13210	14931	19022
496	50.63	31.54	3858	4819	5340	6075	6658	7268	8567	10218	11749	13386	15131	19276
510	52.03	31.97	3912	4887	5415	6161	6752	7370	8687	10361	11913	13574	15343	19547
524	53.44	32.71	3965	4953	5488	6245	6844	7470	8806	10503	12076	13759	15552	19813
538	54.85	32.82	4018	5019	5561	6327	6934	7569	8923	10642	12236	13942	15758	20076
552	56.25	33.25	4070	5084	5633	6409	7024	7667	9038	10780	12394	14122	15962	20335
565	57.66	33.66	4118	5143	5699	6484	7106	7757	9144	10906	12539	14287	16149	20573
579	59.07	34.06	4168	5207	5769	6564	7194	7853	9256	11040	12694	14463	16348	20827
593	60.47	34.47	4218	5269	5839	6643	7280	7947	9368	11173	12846	14637	16544	21077
607	61.88	34.87	4268	5331	5907	6721	7366	8040	9478	11304	12997	14809	16/38	21324
620	64.60	35.26	4313	5388	5970	6960	7590	8120	9578	11424	13130	14900	10917	21552
034	04.09 66 10	35.05	4302	5448	6109	6044	7528	0217	9080	11002	19490	15134	17107	21/94
048	00.10	30.04	4410	5508	0103	0944	7010	0007	9792	110/9	15429	15301	17294	22033

Table 4.10.1(b) Theoretical Discharge Through Circular Orifices (Liters of Water per Minute)

Table 4.10.1(b) Continued

Ditot		X-lit-						Orific	ce Size	(mm)				
Pressure* (kPa)	Meters†	Discharge (m/sec)	51	57	60	64	67	70	76	83	89	95	101	114
662	67.50	36.42	4457	5567	6169	7019	7692	8397	9898	11805	13573	15465	17480	22270
676	68.91	36.79	4504	5626	6234	7093	7773	8485	10002	11929	13716	15628	17664	22504
689	70.32	37.17	4547	5680	6293	7161	7848	8566	10097	12043	13847	15777	17833	22719
703	71.72	37.54	4593	5737	6357	7233	7927	8653	10200	12165	13987	15937	18013	22949
717	73.13	37.90	4638	5794	6420	7305	8005	8738	10301	12285	14126	16095	18192	23176
731	74.54	38.27	4684	5850	6482	7376	8083	8823	10401	12405	14263	16251	18369	23401
745	75.94	38.63	4728	5906	6544	7446	8160	8907	10500	12523	14399	16406	18544	23624
758	77.35	38.98	4769	5957	6601	7510	8231	8985	10591	12632	14524	16548	18705	23830
772	78.76	39.33	4813	6012	6662	7580	8307	9067	10688	12748	14658	16701	18877	24049
786	80.16	39.68	4857	6066	6722	7648	8382	9149	10785	12863	14790	16851	19047	24266
800	81.57	40.03	4900	6120	6781	7716	8456	9230	10880	12977	14921	17001	19216	24481
813	82.97	40.37	4939	6170	6836	7778	8525	9305	10968	13082	15042	17138	19371	24679
827	84.38	40.71	4982	6223	6895	7845	8598	9385	11063	13194	15171	17285	19538	24891
841	85.79	41.05	5024	6275	6953	7911	8670	9464	11156	13305	15299	17431	19702	25100
855	87.19	41.39	5065	6327	7011	7977	8742	9542	11248	13416	15425	17575	19866	25309
869	88.60	41.72	5107	6379	7068	8042	8813	9620	11340	13525	15551	17719	20028	25515
882	90.01	42.05	5145	6426	7121	8102	8879	9692	11424	13626	15667	17851	20177	25705
896	91.41	42.38	5185	6477	7177	8166	8949	9768	11515	13734	15791	17992	20336	25908
910	92.82	42.70	5226	6527	7233	8229	9019	9844	11604	13840	15914	18132	20495	26110
924	94.23	43.03	5266	6577	7288	8292	9088	9920	11693	13947	16036	18271	20652	26310
938	95.63	43.35	5305	6627	7343	8355	9156	9995	11782	14052	16157	18409	20807	26509

Notes:

(1) This table is computed from the formula $Q_m = 0.0666cd^2\sqrt{P_m}$, with c = 1.00. The theoretical discharge of seawater, as from fireboat nozzles, can be found by subtracting 1 percent from the figures in Table 4.10.2.1, or from the formula $Q_m = 0.065cd^2m\sqrt{P_m}$.

(2) Appropriate coefficient should be applied where it is read from the hydrant outlet. Where more accurate results are required, a coefficient appropriate on the particular nozzle must be selected and applied to the figures of the table. The discharge from circular openings of sizes other than those in the table can readily be computed by applying the principle that quantity discharged under a given head varies as the square of the diameter of the opening.

*This pressure corresponds to velocity head.

 $\pm 1 \text{ kPa} = 0.102 \text{ m}$ of water. For pressure in bars, multiply by 0.01.

4.10.1.1 If more than one outlet is used, the discharges from all are added to obtain the total discharge.

4.10.1.2 The formula that is generally used to compute the discharge at the specified residual pressure or for any desired pressure drop is Equation 4.10.1.2:

$$Q_{R} = Q_{F} \times \frac{h_{r}^{0.54}}{h_{f}^{0.54}}$$
(4.10.1.2)

where:

 Q_R = flow predicted at desired residual pressure

 \tilde{Q}_F = total flow measured during test

 h_r = pressure drop to desired residual pressure

 h_f = pressure drop measured during test

4.10.1.3 In this equation, any units of discharge or pressure drop may be used as long as the same units are used for each value of the same variable.

4.10.1.4 In other words, if Q_R is expressed in gpm, Q_F must be in gpm, and if h_r is expressed in psi, h_f must be expressed in psi.

4.10.1.5 These are the units that are normally used in applying Equation 4.10.1.2 to fire flow test computations.

4.10.2 Discharge Calculations from Table.

4.10.2.1 One means of solving this equation without the use of logarithms is by using Table 4.10.2.1, which gives the values of the 0.54 power of the numbers from 1 to 175.

4.10.2.2 Knowing the values of h_f , h_r , and Q_F , the values of $h_f^{0.54}$ and $h_r^{0.54}$ can be read from the table and Equation 4.10.1.2 solved for Q_{fe}

4.10.2.3 Results are usually carried to the nearest 100 gpm (380 L/min) for discharges of 1000 gpm (3800 L/min) or more, and to the nearest 50 gpm (190 L/min) for smaller discharges, which is as close as can be justified by the degree of accuracy of the field observations.

4.10.2.4 Insert in Equation 4.10.1.2 the values of $h_r^{0.54}$ and $h_f^{0.54}$ determined from the table and the value of Q_E , and solve the equation for Q_R .

4.11 Data Sheet.

4.11.1 The data secured during the testing of hydrants for uniform marking can be valuable for other purposes.

h	$h^{0.54}$	h	$h^{0.54}$	h	$h^{0.54}$	h	$h^{0.54}$	h	$h^{0.54}$
1	1.00	36	6.93	71	9.99	106	12.41	141	14.47
2	1.45	37	7.03	72	10.07	107	12.47	142	14.53
3	1.81	38	7.13	73	10.14	108	12.53	143	14.58
4	2.11	39	7.23	74	10.22	109	12.60	144	14.64
5	2.39	40	7.33	75	10.29	110	12.66	145	14.69
6	2.63	41	7.43	76	10.37	111	12.72	146	14.75
7	2.86	42	7.53	77	10.44	112	12.78	147	14.80
8	3.07	43	7.62	78	10.51	113	12.84	148	14.86
9	3.28	44	7.72	79	10.59	114	12.90	149	14.91
10	3.47	45	7.81	80	10.66	115	12.96	150	14.97
11	3.65	46	7.91	81	10.73	116	13.03	151	15.02
12	3.83	47	8.00	82	10.80	117	13.09	152	15.07
13	4.00	48	8.09	83	10.87	118	13.15	153	15.13
14	4.16	49	8.18	84	10.94	119	13.21	154	15.18
15	4.32	50	8.27	85	11.01	120	13.27	155	15.23
16	4.48	51	8.36	86	11.08	121	13.33	156	15.29
17	4.62	52	8.44	87	11.15	122	13.39	157	15.34
18	4.76	53	8.53	88	11.22	123	13.44	158	15.39
19	4.90	54	8.62	89	11.29	124	13.50	159	15.44
20	5.04	55	8.71	90	11.36	125	13.56	160	15.50
21	5.18	56	8.79	91	11.43	126	13.62	161	15.55
22	5.31	57	8.88	92	11.49	127	13.68	162	15.60
23	5.44	58	8.96	93	11.56	128	13.74	163	15.65
24	5.56	59	9.04	94	11.63	129	13.80	164	15.70
25	5.69	60	9.12	95	11.69	130	13.85	165	15.76
26	5.81	61	9.21	96	11.76	131	13.91	166	15.81
27	5.93	62	9.29	97	11.83	132	13.97	167	15.86
28	6.05	63	9.37	98	11.89	133	14.02	168	15.91
29	6.16	64	9.45	99	11.96	134	14.08	169	15.96
30	6.28	65	9.53	100	12.02	135	14.14	170	16.01
31	6.39	66	9.61	101	12.09	136	14.19	171	16.06
32	6.50	67	9.69	102	12.15	137	14.25	172	16.11
33	6.61	68	9.76	103	12.22	138	14.31	173	16.16
34	6.71	69	9.84	104	12.28	139	14.36	174	16.21
35	6.82	70	9.92	105	12.34	140	14.42	175	16.26

Table 4.10.2.1 Values of h to the 0.54 Power

4.11.2 With this in mind, it is suggested that the form shown in Figure 4.11.2 be used to record information that is taken.

4.11.3 The back of the form should include a location sketch.

4.11.4 Results of the flow test should be indicated on a hydraulic graph, such as the one shown in Figure 4.11.4.

4.11.5 When the tests are complete, the forms should be filed for future reference by interested parties.

4.12 System Corrections.

4.12.1 It must be remembered that flow test results show the strength of the distribution system and do not necessarily indicate the degree of adequacy of the entire water works system.

4.12.2 Consider a system supplied by pumps at one location and having no elevated storage.

4.12.3 If the pressure at the pump station drops during the test, it is an indication that the distribution system is capable of delivering more than the pumps can deliver at their normal operating pressure.

4.12.4 It is necessary to use a value for the drop in pressure for the test that is equal to the actual drop obtained in the field during the test, minus the drop in discharge pressure at the pumping station.

4.12.5 If sufficient pumping capacity is available at the station and the discharge pressure could be maintained by operating additional pumps, the water system as a whole could deliver the computed quantity.

4.12.6 If, however, additional pumping units are not available, the distribution system would be capable of delivering the computed quantity, but the water system as a whole would be limited by the pumping capacity.

4.12.7 The portion of the pressure drop for which a correction can be made for tests on systems with storage is generally estimated upon the basis of a study of all the tests made and the pressure drops observed on the recording gauge at the station for each.

4.12.8 The corrections may vary from very substantial portions of the observed pressure drops for tests near the pumping station, to zero for tests remote from the station.



	Hydrant I	Flow Test F	Report		
Location				Date	
Test made by				Time	
Representative of					
Witness					
State purpose of test					
Consumption rate during If pumps affect test, indi Flow hydrants: Size nozzle Pitot reading Discharge coefficient gpm Static B Projected results @20 Remarks	g test cate pumps o A ₁ psi psi Residual_	A2 Residua	A ₃	A ₄ Tota si Residual_	_ gpm _ psi _ gpm
Location map: Show line siz hydrant branch size. Indi location of static and res Indicate B Hydrant	es and distance cate north. Shov dual – Label B. . Sprinkler	to next cross v flowing hyd Other (id	s-connected rants – Labe entify)	I line. Show valve el A ₁ , A ₂ , A ₃ , A ₄ .	s and Show

FIGURE 4.11.2 Sample Report of a Hydrant Flow Test.

Chapter 5 Marking of Hydrants

5.1 Classification of Hydrants. Hydrants should be classified in accordance with their rated capacities [at 20 psi (1.4 bar) residual pressure or other designated value] as follows:

- (1) Class AA Rated capacity of 1500 gpm (5680 L/min) or greater
- (2) Class A Rated capacity of 1000–1499 gpm (3785–5675 L/min)
- (3) Class B Rated capacity of 500–999 gpm (1900–3780 L/min)
- (4) Class C Rated capacity of less than 500 gpm (1900 L/min)

5.2 Marking of Hydrants.

5.2.1 Public Hydrants.

5.2.1.1 All barrels are to be chrome yellow except in cases where another color has already been adopted.

5.2.1.2 The tops and nozzle caps should be painted with the following capacity-indicating color scheme to provide simplicity and consistency with colors used in signal work for safety, danger, and intermediate condition:

- (1) Class AA Light blue
- (2) Class A Green
- (3) Class B Orange
- (4) Class C Red

5.2.1.3 For rapid identification at night, it is recommended that the capacity colors be of a reflective-type paint.



FIGURE 4.11.4 Sample Graph Sheet.

5.2.1.4 Hydrants rated at less than 20 psi (1.4 bar) should have the rated pressure stenciled in black on the hydrant top.

5.2.1.5 In addition to the painted top and nozzle caps, it can be advantageous to stencil the rated capacity of high-volume hydrants on the top.

5.2.1.6 The classification and marking of hydrants provided for in this chapter anticipate determination based on individual flow test.

5.2.1.7 Where a group of hydrants can be used at the time of a fire, some special marking designating group-flow capacity may be desirable.

5.2.2 Permanently Inoperative Hydrants. Fire hydrants that are permanently inoperative or unusable should be removed.

5.2.3 Temporarily Inoperative Hydrants. Fire hydrants that are temporarily inoperative or unusable should be wrapped or otherwise provided with temporary indication of their condition.

5.2.4 Flush Hydrants. Location markers for flush hydrants should carry the same background color as stated above for class indication, with such other data stenciled thereon as deemed necessary.

5.2.5 Private Hydrants. Marking on private hydrants within private enclosures is to be at the owner's discretion. When private hydrants are located on public streets, they should be painted red, or some other color, to distinguish them from public hydrants.

Annex A Explanatory Material

Annex A is not a part of the recommendations of this NFPA document but is included for informational purposes only. This annex contains explanatory material, numbered to correspond with the applicable text paragraphs.

A.3.2.1 Authority Having Jurisdiction (AHJ). The phrase "authority having jurisdiction," or its acronym AHJ, is used in NFPA documents in a broad manner, since jurisdictions and approval agencies vary, as do their responsibilities. Where public safety is primary, the authority having jurisdiction may be a federal, state, local, or other regional department or individual such as a fire chief; fire marshal; chief of a fire prevention bureau, labor department, or health department; building official; electrical inspector; or others having statutory authority. For insurance purposes, an insurance inspection department, rating bureau, or other insurance company representative may be the authority having jurisdiction. In many circumstances, the property owner or his or her designated agent assumes the role of the authority having jurisdiction; at government installations, the commanding officer or departmental official may be the authority having jurisdiction.

A.3.2.2 Listed. The means for identifying listed equipment may vary for each organization concerned with product evaluation; some organizations do not recognize equipment as listed unless it is also labeled. The authority having jurisdiction should utilize the system employed by the listing organization to identify a listed product.

Annex B Informational References (Reserved)

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Sequence of Events Leading to Issuance of an NFPA Committee Document

Step 1: Call for Proposals

•Proposed new Document or new edition of an existing Document is entered into one of two yearly revision cycles, and a Call for Proposals is published.

Step 2: Report on Proposals (ROP)

- •Committee meets to act on Proposals, to develop its own Proposals, and to prepare its Report.
- •Committee votes by written ballot on Proposals. If twothirds approve, Report goes forward. Lacking two-thirds approval, Report returns to Committee.
- •Report on Proposals (ROP) is published for public review and comment.

Step 3: Report on Comments (ROC)

- •Committee meets to act on Public Comments to develop its own Comments, and to prepare its report.
- •Committee votes by written ballot on Comments. If twothirds approve, Report goes forward. Lacking two-thirds approval, Report returns to Committee.
- •Report on Comments (ROC) is published for public review.

Step 4: Technical Report Session

- "Notices of intent to make a motion" are filed, are reviewed, and valid motions are certified for presentation at the Technical Report Session. ("Consent Documents" that have no certified motions bypass the Technical Report Session and proceed to the Standards Council for issuance.)
- •NFPA membership meets each June at the Annual Meeting Technical Report Session and acts on Technical Committee Reports (ROP and ROC) for Documents with "certified amending motions."
- •Committee(s) vote on any amendments to Report approved at NFPA Annual Membership Meeting.

Step 5: Standards Council Issuance

- •Notification of intent to file an appeal to the Standards Council on Association action must be filed within 20 days of the NFPA Annual Membership Meeting.
- •Standards Council decides, based on all evidence, whether or not to issue Document or to take other action, including hearing any appeals.

Committee Membership Classifications

The following classifications apply to Technical Committee members and represent their principal interest in the activity of the committee.

- M *Manufacturer*: A representative of a maker or marketer of a product, assembly, or system, or portion thereof, that is affected by the standard.
- U *User*: A representative of an entity that is subject to the provisions of the standard or that voluntarily uses the standard.
- I/M *Installer/Maintainer:* A representative of an entity that is in the business of installing or maintaining a product, assembly, or system affected by the standard.
- L *Labor:* A labor representative or employee concerned with safety in the workplace.
- R/T Applied Research/Testing Laboratory: A representative of an independent testing laboratory or independent applied research organization that promulgates and/or enforces standards.
- E *Enforcing Authority:* A representative of an agency or an organization that promulgates and/or enforces standards.
- I *Insurance:* A representative of an insurance company, broker, agent, bureau, or inspection agency.
- C *Consumer:* A person who is, or represents, the ultimate purchaser of a product, system, or service affected by the standard, but who is not included in the *User* classification.
- SE *Special Expert:* A person not representing any of the previous classifications, but who has a special expertise in the scope of the standard or portion thereof.

NOTES;

1. "Standard" connotes code, standard, recommended practice, or guide.

2. A representative includes an employee.

3. While these classifications will be used by the Standards Council to achieve a balance for Technical Committees, the Standards Council may determine that new classifications of members or unique interests need representation in order to foster the best possible committee deliberations on any project. In this connection, the Standards Council may make appointments as it deems appropriate in the public interest, such as the classification of "Utilities" in the National Electrical Code Committee.

4. Representatives of subsidiaries of any group are generally considered to have the same classification as the parent organization.

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